

Non-convex Gao model for nonlinear elastic beam with inclined non-penetrating crack

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Abstract

Large deflections of an elastic beam are considered within Euler–Bernoulli hypotheses. The Gao beam model couples nonlinear constitutive equations for longitudinal and vertical displacements. The setting is motivated to describe non-penetration between surfaces of an inclined crack in the thick beam. The governing relations are established for the cracked beam, which are based on constrained minimization of a non-convex function of the strain energy. The variational solution is proved, optimality conditions are obtained in the form of variational inequality, and the complete system of nonlinear boundary conditions fulfilled at the crack is derived.

Keywords

Nonlinear beam, crack, non-penetration condition, non-convex optimization, variational inequality

1. Introduction

In this contribution, we formulate a coupled model which describes large vertical displacement (deflection) together with horizontal (longitudinal) displacement in a thick elastic beam based on the Euler–Bernoulli distribution. The nonlinear constitutive relations keep the second-order asymptotic terms and are stated in plane stress conditions according to Gao [1]. The basic theory of nonlinear beams, plates, and shells can be found in the monographs by Ciarlet and Rabier [2], Timoshenko [3] and von Kármán [4], and suitable variational and dual principles by Gao [5], Khludnev and Kovtunenکو [6], and Khludnev and Sokolowski [7]. For analysis of the respective obstacle problems, we refer to Ciarlet et al. [8], Ciarlet and

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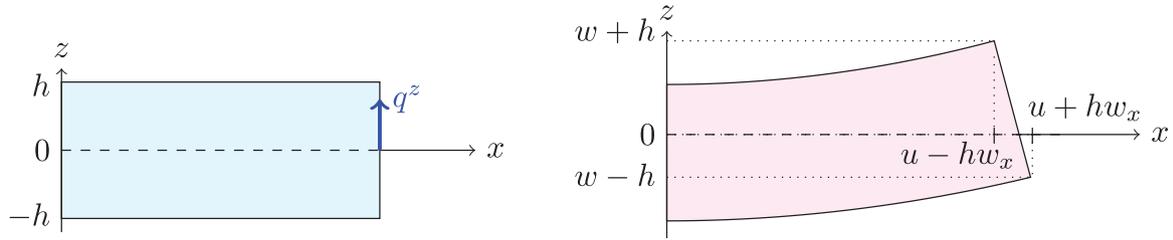


Figure 1. Undeformed (left), and deformed (right) element of thick beam at reference point x .

Piersanti [9], to Lagnese and Leugering [10] and Perla Menzala and Zuazua [11] for parameter sensitivity in beam dynamics. One of the principal difficulties of the nonlinear theory concerns so-called buckling deformation which leads to non-convex energy functions and, as the consequence, to non-uniqueness of solutions, see Russell and White [12]. For semi-coerciveness issues, we cite Goeleven and Gwinner [13].

Gao's model of nonlinear elastic beams is typically formulated as a fourth order semi-linear equation written with respect to vertical deflections referred to the mid-line of the beam. Existence of unique variational solution was provided within minimization of the strain energy under convexity assumption, this prohibiting buckling, in Machalová and Netuka [14]. For modeling with dynamic equations of motion we refer the reader to Andrews et al. [15], for a singular equation under moving point-wise load to Atlasiuk et al. [16] and Bauer et al. [17], for inverse parameter identification problem to Radová et al. [18], and for numerical solution to Borsos and Karátson [19]. An extended Gao's model coupling nonlinear constitutive equations for large deformation within Timoshenko hypotheses was presented in Dyniewicz et al. [20]. Simulations of a contact problem for a nonlinear dynamic beam with a crack were described in Kuttler et al. [21].

In our modeling, we are motivated by non-penetration conditions between opposite surfaces of an inclined crack (the cross cut) in the thick beam. For this task, we utilize both horizontal and vertical displacements along the beam thickness. The variational theory of elastic and inelastic bodies with non-penetrating cracks is well established, see Bach et al. [22], Hintermüller et al. [23], Itou et al. [24], and Khudnev et al. [25], numerical simulation in Kovtunenکو [26], and a review on edge crack problems in Singh [27]. Inequality constraints subjected to non-penetration across inclined cracks were first introduced in Kovtunenکو et al. [28] and developed further in works by Lazarev and Popova [29] and Rudoy and Lazarev [30]. Variational inequalities describing non-penetrating cracks were treated in Kovtunenکو and Lazarev [31] for the Timoshenko plate contacting at the boundary with an inclined obstacle. We cite relevant obstacle problems over non-convex feasible sets in Lazarev and Kovtunenکو [32, 33]. See generalization of inequality constraints to global injectivity conditions for rigid inclusions and cracks in Furtsev et al. [34].

Within the energy minimization approach, we consider the strain energy function for the Gao beam subject to non-penetration inequality imposed on an inclined crack. The function is non-quadratic and non-convex allowing buckling. We provide a variational solution to the problem, which may be non-unique, by the direct method in the calculus of variations owing to coercivity and weak lower semi-continuity properties of the energy function. Due to the lack of convexity of the strain energy, the necessary optimality condition is derived in the form of a variational inequality over convex closed set of feasible displacements. A complete system of nonlinear relations which consist of equilibrium equations and boundary conditions fulfilled at the crack is proved rigorously.

2. Modeling of nonlinear beam

Let an elastic beam of the length $L > 0$ and uniform half-thickness $h > 0$ before deformation occupy the rectangle $(0, L) \times (-h, h)$ in the (x, z) -cross section drawn in Figure 1 (left). After deformation under applied vertical load $q^z(x)$, its longitudinal and vertical displacement $\mathbf{u}^z = (u^z, w^z)(x)$ with rotation angle $w_x := \partial w / \partial x$ to perpendicular cross section are given by Euler–Bernoulli hypotheses:

$$u^z := u - zw_x, \quad w^z := w, \quad (1)$$

where $\mathbf{u} = (u, w)(x)$ are displacement components in the mid-line $z=0$ of the beam, see illustration of the deformed configuration in Figure 1 (right). The Green–St Venant finite strain tensor $\varepsilon^z := (\nabla \mathbf{u}^z + (\nabla \mathbf{u}^z)^\top + (\nabla \mathbf{u}^z)^\top \nabla \mathbf{u}^z)/2$ is computed from equation (1) as follows:

$$\varepsilon^z = \begin{pmatrix} u_x - zw_{xx} + \frac{1}{2}(u_x - zw_{xx})^2 + \frac{1}{2}w_x^2, & -\frac{1}{2}(u_x - zw_{xx})w_x \\ -\frac{1}{2}(u_x - zw_{xx})w_x, & \frac{1}{2}w_x^2 \end{pmatrix}. \quad (2)$$

Here, w_{xx} implies the curvature. Assume the asymptotic order $w \sim h/L$ and $u \sim \epsilon$ such that $w_x \sim \epsilon$, $u_x \sim \epsilon^2$, $w_{xx} \sim \epsilon^2$. Omitting the cubic order asymptotic terms in equation (2) the strain is approximated by

$$\varepsilon^z \sim \varepsilon(\mathbf{u}) - z \begin{pmatrix} w_{xx}, & 0 \\ 0, & 0 \end{pmatrix}, \quad \varepsilon(\mathbf{u}) := \begin{pmatrix} u_x + \frac{1}{2}w_x^2, & 0 \\ 0, & \frac{1}{2}w_x^2 \end{pmatrix}. \quad (3)$$

For Poisson's ratio $\nu \in (-1, 0.5)$, Young's modulus $E > 0$, and shear modulus $G = E/(2(1 + \nu))$, following Gao [1] we assume the plain stress $\sigma^z := 2G\varepsilon^z + E\nu/(1 - \nu^2)(\text{tr}\varepsilon^z)\mathbf{I}$, where \mathbf{I} denotes the identity tensor, such that $\sigma^z \sim \sigma(\mathbf{u}) - \frac{Ez}{1 - \nu^2} \begin{pmatrix} w_{xx}, & 0 \\ 0, & \nu w_{xx} \end{pmatrix}$ and

$$\sigma(\mathbf{u}) := \frac{E}{1 - \nu^2} \begin{pmatrix} u_x + \frac{1 + \nu}{2}w_x^2, & 0 \\ 0, & \nu u_x + \frac{1 + \nu}{2}w_x^2 \end{pmatrix}. \quad (4)$$

Integrating over the beam thickness with the use of identity $\int_{-h}^h z dz = 0$ we get the integrated stress:

$$\mathbf{N}(\mathbf{u}) := \int_{-h}^h \sigma(\mathbf{u}) dz = \frac{2Eh}{1 - \nu^2} \begin{pmatrix} u_x + \frac{1 + \nu}{2}w_x^2, & 0 \\ 0, & \nu u_x + \frac{1 + \nu}{2}w_x^2 \end{pmatrix}, \quad (5)$$

and the bending moment:

$$M(w) := \frac{E}{1 - \nu^2} \int_{-h}^h w_{xx}z^2 dz = \frac{2Eh^3}{3(1 - \nu^2)} w_{xx}. \quad (6)$$

We introduce the beam energy $\Pi^z := \int_0^L \int_{-h}^h (1/2\sigma^z : \varepsilon^z - q^z w^z) dz dx$ using the double contraction of tensors, which after substitution of equations (3)–(6) is approximated as $\Pi^z \sim \Pi(\mathbf{u})$ with $q := \int_{-h}^h q^z dz$ and

$$\begin{aligned} \Pi(\mathbf{u}) &:= \int_0^L \left(\frac{1}{2}\mathbf{N}(\mathbf{u}) : \varepsilon(\mathbf{u}) + \frac{1}{2}M(w)w_{xx} - qw \right) dx \\ &= \int_0^L \left(\frac{Eh}{1 - \nu^2} \left((u_x + \frac{1 + \nu}{2}w_x^2)^2 + \frac{1 - \nu^2}{4}w_x^4 + \frac{h^2}{3}w_{xx}^2 \right) - qw \right) dx. \end{aligned} \quad (7)$$

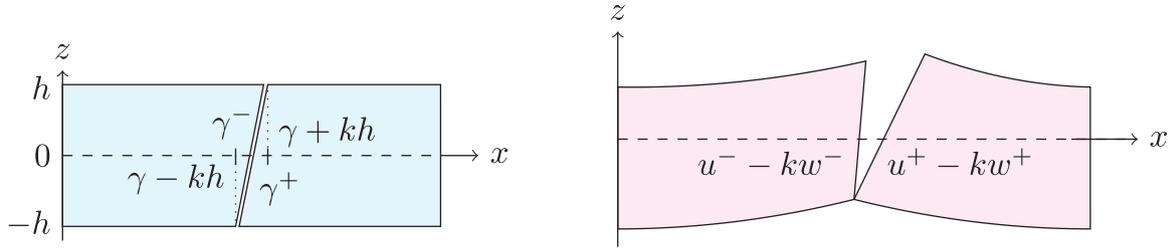


Figure 2. Undeformed (left), and deformed (right) element of beam with inclined crack at $x = \gamma$ ($k > 0$).

Varying equation (7) with test functions $\bar{\mathbf{u}} = (\bar{u}, \bar{w})(x)$:

$$\int_0^L \frac{2Eh}{1-\nu^2} \left(u_x + \frac{1+\nu}{2} w_x^2 \right) \bar{u}_x dx = 0,$$

$$\int_0^L \left[\frac{2Eh}{1-\nu^2} \left(\left(u_x + \frac{1+\nu}{2} w_x^2 \right) (1+\nu) w_x + \frac{1-\nu^2}{2} w_x^3 \right) \bar{w}_x + \frac{h^2}{3} w_{xx} \bar{w}_{xx} \right] - q \bar{w} dx = 0, \quad (8)$$

integrating by parts when $\bar{\mathbf{u}}(0) = \bar{\mathbf{u}}(L) = 0$, $w_x(0) = w_x(L) = 0$, and using flexural rigidity $D = 2Eh^3 / (3(1 - \nu^2))$ follows equilibrium equations for $x \in (0, L)$:

$$\frac{3D}{h^2} \left(u_x + \frac{1+\nu}{2} w_x^2 \right)_x = 0, \quad \frac{3D}{h^2} \left(- \left(\left(u_x + \frac{1+\nu}{2} w_x^2 \right) (1+\nu) w_x \right)_x - \frac{1-\nu^2}{2} (w_x^3)_x + \frac{h^2}{3} w_{xxxx} \right) - q = 0. \quad (9)$$

Let us introduce constant p to reduce equation (9) to the coupled Gao equations:

$$4Gh \left(u_x + \frac{1+\nu}{2} w_x^2 \right) = p, \quad Dw_{xxxx} - pw_{xx} - Eh(w_x^3)_x = q.$$

Note that omitting here the nonlinear term $(w_x^3)_x$ follows well-known equations von Kármán, see Lagnese and Leugering [10] and Perla Menzala and Zuazua [11], whereas setting $p = 0$ justifies the fourth order Euler–Bernoulli equation for large deflections.

3. Modeling of inclined non-penetrating crack

Let an interior point $\gamma \in (0, L)$ at the beam mid-line associate a crack (the cross cut of the beam) build by the incline line, which is prescribed by the inverse slope k at $x = \gamma$ before deformation:

$$x - \gamma = kz \quad \text{for } |z| \leq h, \quad |kh| < \min(\gamma, L - \gamma), \quad (10)$$

drawn in Figure 2 (left). After deformation illustrated in Figure 2 (right), according to hypotheses (1) the left crack face γ^- is described by:

$$x - \gamma + u^- - zw_x^- = k(z + w^-) \quad \text{for } |z| \leq h, \quad \mathbf{u}^- := \lim_{s \rightarrow 0, s > 0} \mathbf{u}(\gamma - s),$$

respectively, the right crack face γ^+ is determined according to equation (10):

$$x - \gamma + u^+ - zw_x^+ = k(z + w^+) \quad \text{for } |z| \leq h, \quad \mathbf{u}^+ := \lim_{s \rightarrow 0, s > 0} \mathbf{u}(\gamma + s).$$

Non-penetration between the crack faces requires the non-negative jump:

$$[[u - zw_x - kw]] \geq 0 \quad \text{for } |z| \leq h, \quad [[\mathbf{u}]] := \mathbf{u}^+ - \mathbf{u}^-.$$

Owing to the linearity in z , this inequality holds for $z = h$ and $z = -h$, which is equivalent to

$$[[u - kw]] \geq h |[[w_x]]| \quad \text{at } x = \gamma. \quad (11)$$

Note that the particular case of $k = 0$ in equation (11) describes the straight crack, see Kovtunenکو et al. [28]. In the following we refer to the beam with the crack in the mid-line $\Omega_\gamma := (0, \gamma^-) \cup (\gamma^+, L)$.

4. Variational problem

Let us denote the Sobolev spaces $H_{0,L}^1(\Omega_\gamma) := \{u \in H^1(\Omega_\gamma) : u(0) = u(L) = 0\}$, and $H_{0,L}^2(\Omega_\gamma) := \{w \in H^2(\Omega_\gamma) : w(0) = w(L) = w_x(0) = w_x(L) = 0\}$. We define the set of admissible functions:

$$\mathbf{K} := \{\mathbf{u} \in \mathbf{V} : \mathbf{u} \text{ satisfies equation (11)}\}, \quad \mathbf{V} := \{\mathbf{u} = (u, w) \in H_{0,L}^1(\Omega_\gamma) \times H_{0,L}^2(\Omega_\gamma)\},$$

which is a convex closed cone. For given $q \in L^2(\Omega_\gamma)$, we determine the strain energy functional $\Pi : \mathbf{V} \mapsto \mathbb{R}$ of (7):

$$\Pi(\mathbf{u}) := \int_{\Omega_\gamma} \left(\frac{1}{2} \mathbf{N}(\mathbf{u}) : \varepsilon(\mathbf{u}) + \frac{1}{2} M(w) w_{xx} - qw \right) dx = \int_{\Omega_\gamma} \left(\frac{3D}{2h^2} \left(u_x + \frac{1+\nu}{2} w_x^2 \right)^2 + \frac{Eh}{4} w_x^4 + \frac{D}{2} w_{xx}^2 - qw \right) dx. \quad (12)$$

It is well-defined due to the continuous embedding estimates for $w \in H^2(\Omega_\gamma)$:

$$\|w_x\|_{L^2(\Omega_\gamma)} \leq \sqrt{L} \|w_x\|_{C(\Omega_\gamma)} \leq \sqrt{L} c_e \|w_{xx}\|_{L^2(\Omega_\gamma)}, \quad c_e > 0. \quad (13)$$

Also, the Poincaré inequality holds for $\mathbf{u} \in \mathbf{V}$:

$$c_P \|u\|_{H^1(\Omega_\gamma)}^2 \leq \|u_x\|_{L^2(\Omega_\gamma)}^2, \quad c_P \|w\|_{H^2(\Omega_\gamma)}^2 \leq \|w_{xx}\|_{L^2(\Omega_\gamma)}^2, \quad c_P > 0. \quad (14)$$

Based on equations (13) and (14), below we establish useful properties of the beam energy functional.

Lemma 1. $\Pi : \mathbf{V} \mapsto \mathbb{R}$ in equation (12) is coercive with the lower estimate:

$$\Pi(\mathbf{u}) \geq c_c (\|u\|_{H^1(\Omega_\gamma)}^2 + \|w\|_{H^2(\Omega_\gamma)}^2) - \|q\|_{L^2(\Omega_\gamma)} \|w\|_{L^2(\Omega_\gamma)}, \quad c_c > 0. \quad (15)$$

Proof. We estimate from below the first term in equation (12) by applying the Cauchy–Schwarz inequality:

$$\int_{\Omega_\gamma} w_x^2 dx \leq \sqrt{L} \int_{\Omega_\gamma} w_x^4 dx,$$

and using Young's inequality with weight $\epsilon \in (0, 1)$:

$$\int_{\Omega_\gamma} \left(u_x + \frac{1+\nu}{2} w_x^2 \right)^2 dx \geq (1-\epsilon) \|u_x\|_{L^2(\Omega_\gamma)}^2 - \left(\frac{1}{\epsilon} - 1 \right) \frac{(1+\nu)^2}{4} \|w_x^2\|_{L^2(\Omega_\gamma)}^2,$$

which yields the lower bound:

$$\begin{aligned} \Pi(\mathbf{u}) \geq \frac{3D}{2h^2} (1-\epsilon) \|u_x\|_{L^2(\Omega_\gamma)}^2 + \frac{Eh}{4} \left(1 - \frac{1}{1-\nu^2} \left(\frac{1}{\epsilon} - 1 \right) \frac{(1+\nu)^2}{L} \right) \|w_x\|_{L^2(\Omega_\gamma)}^4 \\ + \frac{D}{2} \|w_{xx}\|_{L^2(\Omega_\gamma)}^2 - \|q\|_{L^2(\Omega_\gamma)} \|w\|_{L^2(\Omega_\gamma)}. \end{aligned}$$

Choosing here $1/(1 + L(1 + \nu)/(1 - \nu)) \leq \epsilon < 1$ together with equations (13) and (14) proves equation (15). \square

Here, it is worth noting that the first, nonlinear term in the energy functional is non-convex allowing buckling deformation, see Machalová and Netuka [14].

Lemma 2. $\Pi: \mathbf{V} \mapsto \mathbb{R}$ in equation (12) is weakly lower semi-continuous:

$$\mathbf{u}^n \rightharpoonup \mathbf{u} \implies \liminf_{n \rightarrow \infty} \Pi(\mathbf{u}^n) \geq \Pi(\mathbf{u}). \quad (16)$$

Proof. Suppose there exists a weakly convergent sequence:

$$\mathbf{u}^n \rightharpoonup \mathbf{u} \quad \text{weakly in } H^1(\Omega_\gamma) \times H^2(\Omega_\gamma) \text{ as } n \rightarrow \infty. \quad (17)$$

The Sobolev compact embedding provides that

$$w_x^n \rightarrow w_x \quad \text{strongly in } L^4(\Omega_\gamma) \text{ as } n \rightarrow \infty. \quad (18)$$

Let us express the first, nonlinear term in $\Pi(\mathbf{u}^n)$ (omitting the factor $3D/(2h^2)$ for short):

$$\begin{aligned} \int_{\Omega_\gamma} \left(u_x^n + \frac{1+\nu}{2} (w_x^n)^2 \right)^2 dx &= \left\| u_x^n + \frac{1+\nu}{2} (w_x^n)^2 - u_x - \frac{1+\nu}{2} w_x^2 \right\|_{L^2(\Omega_\gamma)}^2 \\ &+ \int_{\Omega_\gamma} \left(u_x + \frac{1+\nu}{2} w_x^2 \right)^2 dx + 2 \int_{\Omega_\gamma} \left(u_x + \frac{1+\nu}{2} w_x^2 \right) \left(u_x^n + \frac{1+\nu}{2} (w_x^n)^2 - u_x - \frac{1+\nu}{2} w_x^2 \right) dx. \end{aligned}$$

With the use of equation (17) we infer that

$$\liminf_{n \rightarrow \infty} \int_{\Omega_\gamma} \left(u_x^n + \frac{1+\nu}{2} (w_x^n)^2 \right)^2 dx \geq \int_{\Omega_\gamma} \left(u_x + \frac{1+\nu}{2} w_x^2 \right)^2 dx. \quad (19)$$

Then equations (17)–(19) guarantee the weak lower semi-continuity property (16). \square

Consider the minimization problem: find $\mathbf{u} \in \mathbf{K}$ such that

$$\Pi(\mathbf{u}) = \min_{\bar{\mathbf{u}} = (\bar{u}, \bar{w}) \in \mathbf{K}} \Pi(\bar{\mathbf{u}}). \quad (20)$$

As a consequence of the direct method in the calculus of variations, from Lemma 1 and Lemma 2, we conclude straightforwardly with the existence theorem.

Theorem 1. *There exists argument of the minimum (20).*

Because of buckling deformation, the solution to equation (20) may be non-unique.

5. Optimality conditions

Based on the variation (8) of the strain energy and equilibrium equations (9), using notation (5) for the integrated stress and equation (6) for the bending moment, we deduce Green's formula in the cracked domain accounting for the jump across γ , see Khludnev and Kovtunenکو [6]:

$$\begin{aligned} \int_{\Omega_\gamma} [N_{11}(\mathbf{u})\bar{u}_x + \text{tr}\mathbf{N}(\mathbf{u})w_x\bar{w}_x + M(w)\bar{w}_{xx} - q\bar{w}] dx \\ = \int_{\Omega_\gamma} [-N_{11}(\mathbf{u})_x\bar{u} + (-\text{tr}\mathbf{N}(\mathbf{u})w_x)_x + M(w)_{xx} - q]\bar{w}] dx \\ + \llbracket -N_{11}(\mathbf{u})\bar{u} + (-\text{tr}\mathbf{N}(\mathbf{u})w_x + M(w)_x)\bar{w} - M(w)\bar{w}_x \rrbracket, \quad (21) \end{aligned}$$

where $\text{tr}\mathbf{N}(\mathbf{u}) := N_{11}(\mathbf{u}) + N_{22}(\mathbf{u})$.

Theorem 2. *The (necessary) optimality condition of equation (20) yields the variational inequality:*

$$\mathbf{u} \in \mathbf{K}, \quad \int_{\Omega_\gamma} \left[N_{11}(\mathbf{u})(\bar{u} - u)_x + \text{tr}\mathbf{N}(\mathbf{u})w_x(\bar{w} - w)_x + M(w)(\bar{w} - w)_{xx} - q(\bar{w} - w) \right] dx \geq 0 \quad \text{for all } \bar{\mathbf{u}} \in \mathbf{K}. \quad (22)$$

If the solution is smooth such that $\mathbf{u} \in H^2(\Omega_\gamma) \times H^4(\Omega_\gamma)$, then it satisfies the equilibrium equations:

$$N_{11}(\mathbf{u})_x = 0, \quad M(w)_{xx} - N_{22}(\mathbf{u})_x w_x - \text{tr}\mathbf{N}(\mathbf{u})w_{xx} = q \quad \text{in } \Omega_\gamma, \quad (23)$$

and boundary conditions at the crack γ :

$$[[N_{11}(\mathbf{u})]] = 0, \quad [[\text{tr}\mathbf{N}(\mathbf{u})w_x - M(w)_x]] = 0, \quad [[M(w)]] = 0, \quad (24)$$

$$N_{11}(\mathbf{u})[[u]] + (\text{tr}\mathbf{N}(\mathbf{u})w_x - M(w)_x)[[w]] + M(w)[[w_x]] = 0, \quad (25)$$

$$kN_{11}(\mathbf{u}) + \text{tr}\mathbf{N}(\mathbf{u})w_x - M(w)_x = 0, \quad (26)$$

$$-hN_{11}(\mathbf{u}) \geq |M(w)|, \quad [[u - kw]] \geq h|[w_x]|. \quad (27)$$

Proof. The variational inequality (22) follows from the argument of minimum in equation (20) on taking the Gâteaux derivative: $\lim_{t \rightarrow 0} (\Pi(\bar{\mathbf{u}} + t(\bar{\mathbf{u}} - \mathbf{u})) - \Pi(\mathbf{u})) / t \geq 0$. Its solution exists thanks to Theorem 1.

For the smooth solution \mathbf{u} , applying Green’s formula (21) tested by functions $\bar{\mathbf{u}} - \mathbf{u} \in C_0^\infty(\Omega_\gamma)^2$ with compact support yields equilibrium equations (23) and the non-negative reminder

$$[-N_{11}(\mathbf{u})(\bar{u} - u) + (-\text{tr}\mathbf{N}(\mathbf{u})w_x + M(w)_x)(\bar{w} - w) - M(w)(\bar{w} - w)_{xx}] \geq 0.$$

Testing this inequality by continuous functions $\bar{\mathbf{u}} - \mathbf{u} \in C_0^\infty(\Omega)^2$ justifies zero jumps in equation (24), further inserting $\bar{\mathbf{u}} = 0$ and $\bar{\mathbf{u}} = 2\mathbf{u}$ splits it into the equation (25) and the inequality

$$\Theta(\mathbf{u}, [[\bar{\mathbf{u}}]]) := -N_{11}(\mathbf{u})[[\bar{u}]] + (-\text{tr}\mathbf{N}(\mathbf{u})w_x + M(w)_x)[[\bar{w}]] - M(w)[[\bar{w}_x]] \geq 0. \quad (28)$$

We look for the representation of Θ with the help of inequality constraints (11), that is

$$\Theta(\mathbf{u}, [[\bar{\mathbf{u}}]]) = a[[\bar{u} - k\bar{w} - h\bar{w}_x]] + b[[\bar{u} - k\bar{w} + h\bar{w}_x]]. \quad (29)$$

Under the solvability condition (26), unknown a and b can be resolved from equations (28) and (29) as follows:

$$a = \frac{1}{2} \left(-N_{11}(\mathbf{u}) + \frac{M(w)}{h} \right), \quad b = \frac{1}{2} \left(-N_{11}(\mathbf{u}) - \frac{M(w)}{h} \right). \quad (30)$$

Then inequalities (11) and (28) necessitates $a \geq 0, b \geq 0$ in equation (30), which justifies equation (27). \square

The argument in the proof of Theorem 2 can be converted to derive the variational inequality (22) from the boundary value problem (23)–(27). The problem is supported by the boundary conditions at $x = 0, L$ imposed in the space \mathbf{V} . We remark that boundary conditions can vary when describing clamp at the beam end, simply supported end, free end of cantilever beam, and so on.

Authors’ note

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